Total number of printed pages-9

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Thowing over a vocatinal plate of length 2014 and wide wide $\frac{2014}{\log N}$

TRANSFER PROCESS ENGINEERING

Paper : FPT 403

Full Marks : 100

grithrot mill buTime : Three hours

The figures in the margin indicate full marks for the questions.

Answer any five questions.

1. (a) State Newton's law of viscosity. What is the S.I. unit of viscosity ? Explain the variation of viscosity with temperature for liquids and gases. $2+1+3=6$

 $\mu^* \in \text{Rrichoinel vector}$, viology Isnoitoin $\lambda + 5 = 0$

SOUTH CHENGE SECTION DIMENSION Contd.

(b) Prove that the parabolic velocity MINT (EQ) distribution of a liquid of density (ρ) flowing over a vertical flat plate of length (L) and width (W) is

$$
v_z = \frac{\rho g \delta^2}{2\mu} \left[1 - \left(\frac{x}{\delta}\right)^2 \right]
$$

where,

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 v_z = Velocity of fluid in the z-direction

- δ = Thickness of the liquid film forming over the flat plate
- $x = Distance of a shell of the liquid$
	- μ = Viscosity of liquid

And hcncc find thc avcragc velocity and mass flow rate of fluid over the plate.

8+3+3=14 Newton's law of viscosity with is

3

2. (a) Derive time-smoothed continuity equation.

(b) Explain time-smoothed velocity profile ncar a wall.

Show that the thickness of the viscous sublayer (y) is $\frac{\partial v}{u^*}$, where

- $v =$ Kinematic viscosity and
- u^* = Frictional velocity $4+5=9$

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ⁱ @ Derive the general form of time-smoolhed Navier-Stoke equation of the turbulent flow in vector form as shown below :

$$
\frac{\delta}{10100} \frac{\delta}{\delta t} (\rho \overline{V}) = -\nabla \overline{\rho} - \left[\nabla \cdot \rho \overline{V} \overline{V}\right] - \left[\nabla \cdot \left(\overline{\tau}^{\nu} + \overline{\tau}^{\prime}\right)\right] + P g
$$

where, the symbols have their usual meanings. 8

3. (a) Define three modes of heat transfer 3 of the way between the inner and the

 η \leq The rate of flow through the sphere.

(b) State Fourier's law of heat conduction. How does thermal conductivity vary with $\frac{1}{2}$ shi temperature for the gases and liquids ? $2+2=4$ conductivity is R=078 w/m°C.

(c) Consider a 3m high, 6m wide and 0'3m thick brick wall whose thermal for a day conductivity is $K = 0.8 w/m^{\circ} C$. On a certain day, the temperatures of the inner and the **19 answered to** outer surfaces of the wall are measured to 10^{100} and 6° C respectively. Determine od of the rate of heat loss through the wall on $t \to 3$ that day. A bas $\int_0^{\infty} \ln \sqrt{u} \, dt = \frac{1}{2}$ 3

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both (d) A hollow sphere, 10cm I.D. (Inner diameter) and 30cm O.D. (Outer diameter) \sim of a material having thermal conductivity, $50W/mK$ is used as a container for a liquid chemical mixture. Its inner and outer surface temperatures are $300^{\circ}C$ and $100^{\circ}C$ tanzu ric respectively. Determine -

 (i) The rate of flow through the sphere.

 (iii) The temperature at a point, a quarter % of the way between the inner and the outer surfaces State Fouries we surfaces. In the mduction.

thermal

4. (a) Consider a $1.2m$ high and $2m$ wide glass window whose thickness is $6m$ and thermal conductivity is $K = 0.78$ w/m^oC ' Determine the steady rate of heat transfer through this glass window and the temperature of its inner surface for a day during which the room is maintained at 24° C while the temperature of the outdoors of bonus is $-5^{\circ}C$. Take the convection heat transfer coefficients on the inner and the outer
surfaces of the window to be no llaw surfaces of the window $h_1 = 10 \frac{w}{m^3} \cdot C$ and $h_2 = 25 \frac{w}{m^2} \cdot C$. 7

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l.

 (a) Show that the rate of heat flow through a composite hollow cylindrical pipe of lcngth (L) consisting of two co-axial layers of thermal conductivities, K_1 and K_2 if hot milk is flowing inside this pipe is

$$
Q = \frac{2\pi L (T_{\infty 1} - T_{\infty 2})}{\frac{1}{h_1 r_1} + \frac{1}{h_2 r_3} + \frac{ln(r_2/r_1)}{K_1}} + \frac{ln(r_3/r_2)}{K_2}}
$$

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where r_1 and r_2 are the inner and outer radii of inside cylinder and $r₃$ is the outer radius of outside cylinder.

 h_1 and h_2 are the convection heat transfer **Coefficients** at the inner and the outer sword surfaces of the pipe. Woll is not

> T_{α_1} and T_{α_2} are the milk temperature and the outside pipe air temperature respectively. $55560 = 1.5156$

> > **DIMENT (2014)** Contd.

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e rod due ic current

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Average flow velocity

 (c) Prove that the temperature distribution at any radius (r) in a solid cylindrical rod idignol to be duc to the passage of an electric current is

$$
T = T_0 + \frac{\dot{q}}{4K}r_0^2 \left[1 - \left(\frac{r}{r_0}\right)^2\right]
$$

where,

- \dot{q} Heat generation within the rod due to thc passage of an electric currcnt.
- T_0 = Surface temperature of the rod
- r_0 = Radius of rod
- 19806 $K =$ Coefficient of thermal conductivity.
-
- 5. (a) Define Nusselt Number ? How will you cxplain the physical significance of the Nusselt number ? $2+3=5$

 (b) What do you mean by Reynold's number? For a flow of fluid over a flat plate, prove that the Reynold's number is

$$
Re = \frac{VL^{(1)} \cos \frac{1}{2} \sin \frac{1}{2} \sin \frac{1}{2} \cos \frac{1}{2} \sin \frac{1}{2} \sin
$$

where, $L =$ Characteristics length $V =$ Average flow velocity $v =$ kinematic viscosity $2+5=7$

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temperature and **STUJETOGENOT**

t.

(c) Consider a $0.6m \times 0.6m$ thin square plate in a room at 30'C. One side of thc platc is drive not maintained at a temperature of 90°C, while the other side is insulated. Determine the ratc of heat transfer from thc plate by natural **EXAMPLE 28 CONVECTION If the plate is vertical.** 8 goiven data : sent of our whereb

The propertics of air at the film temperature (T_f) of 60°C and latm are:

$$
K = 0.02808 \, \text{w/m}^{\circ} \, C
$$

and the Prandtl Number, $P_r = 0.7202$

 $\frac{1610 \text{ m}}{1610 \text{ m}^2}$ Kinematic viscosity, $v = 1.896 \times 10^{-15} \text{ m}^2/\text{s}$

the inner surface, is defined to be
to not zero. If
$$
\mathbf{B} = \mathbf{A}
$$
 is an arbitrary
independent and the noise of \mathbf{r} at the outer surface.

Nusselt Number for natural convection,

0'387 Ratf.6 | , *1 o.qgzlnlul't" l\rr) l

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6. (a) State Fick's law of mass diffusion. 2

(b) A mixture of oxygen and nitrogen with od online their partial pressure in the ratio 0.21 and 0.79 is in a container at 25° C.

Calculate the molar concentration. mass density, mole fraction and mass fraction of each species for the total pressure of $1 bar.$ Take, $R = 8.314 kJ/kmol.K$. 8

(c) Pressurised hydrogen gas is stored at 358K in a $4.8m$ outer diameter spherical container made of nickel. The shell of the container is 6cm thick. The molar concentration of hydrogen in the nickel at the inner surface is determined to be 0.087 Kmol/m³. The concentration of hydrogen in the nickel at the outer surface is negligible. Determine the mass flow rate ofhydrogen by diffusion through the nickel container if the binary diffusion coefficient for hydrogen in the nickel at the specified temperature is $1.2 \times 10^{-12} m^2/s$. 5

iaquecristics length

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 (d) How does the mass diffusivity of a gas mixture changes with temperature and pressure ?

> Prove that the diffusivities D_{AB} is equal to D_{BA} for an ideal gas where 'A' and 'B' are spccies. $2+3=5$

- 7. Write short notes on **any four** of the follolwing: $5 \times 4 = 20$
	- (a) Importance of transport phenomena in food process engineering
	- Thermal boundary layer on a flat plate (b)
	- Grashof Number (c)
	- Analogy bctween heat transfer and mass transfer (d)
	- Boundary conditions for mass transfer. (e)

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